



GT2500-BRUSH GENERATOR DRIVE TRAIN

Modeling and Analysis

DYNAMICS R4

Alfa-Tranzit Co., Ltd



Customer - Magellan Aerospace, Canada



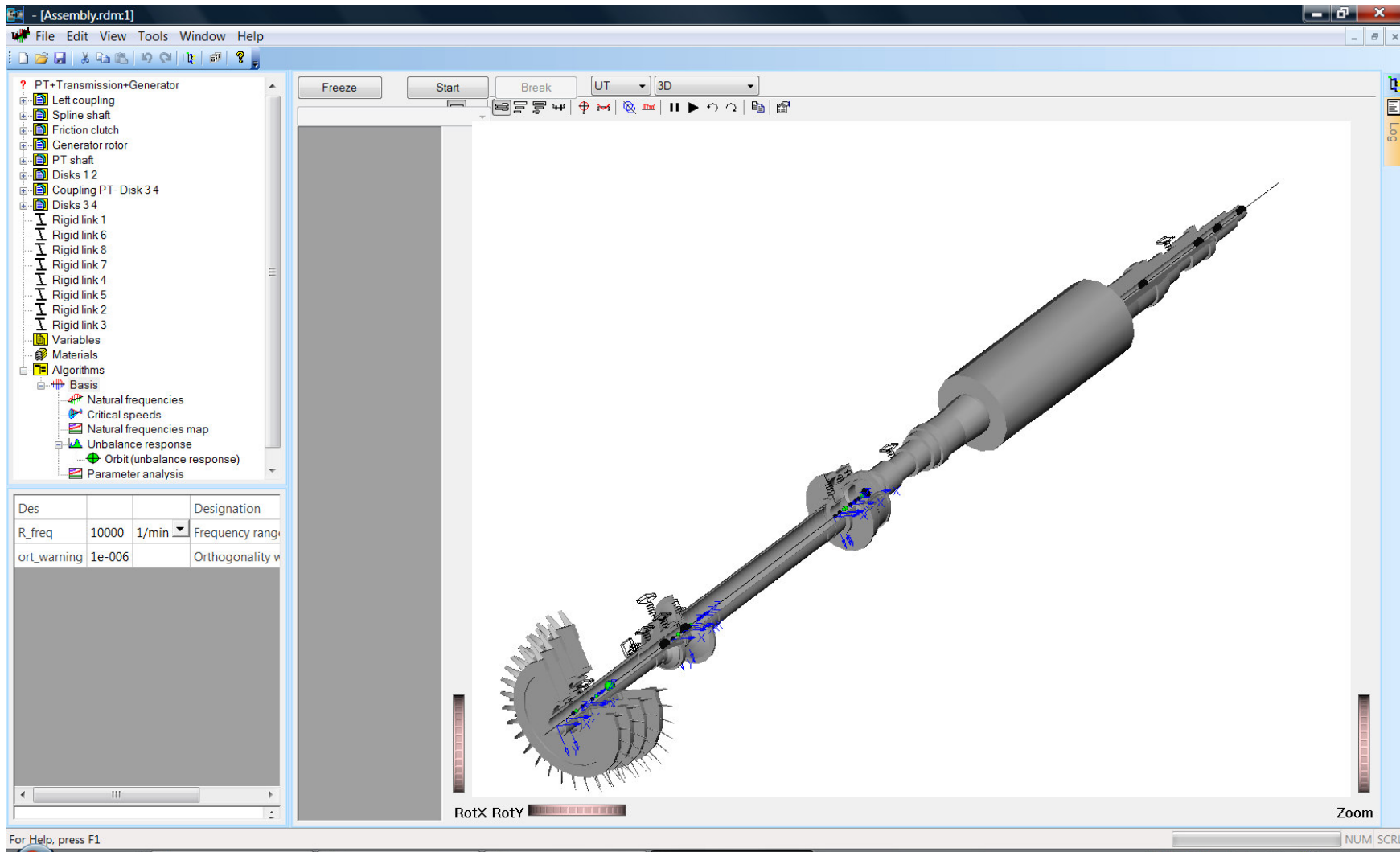
The task of the work is the determination of main resonant modes of the rotor drive train system "GT25000 Power turbine – BRUSH generator" regarding all oscillation modes - bending, torsion and longitudinal ones.

The following was done to accomplish this task:

- the rotor system model creation;
- geometric and mass identification of the rotor system;
- calculation of natural frequencies and mode shapes of all types;
- calculation of critical speeds and the corresponding mode shapes;
- calculation of forced oscillation under the influence of the distributed system of unbalances.

Modeling was carried out in the Dynamics R4.5 program system by the company Alfa-Tranzit Co., Ltd on the basis of the data provided by Magellan Aerospace

Dynamics R4 main window with rotor drive train



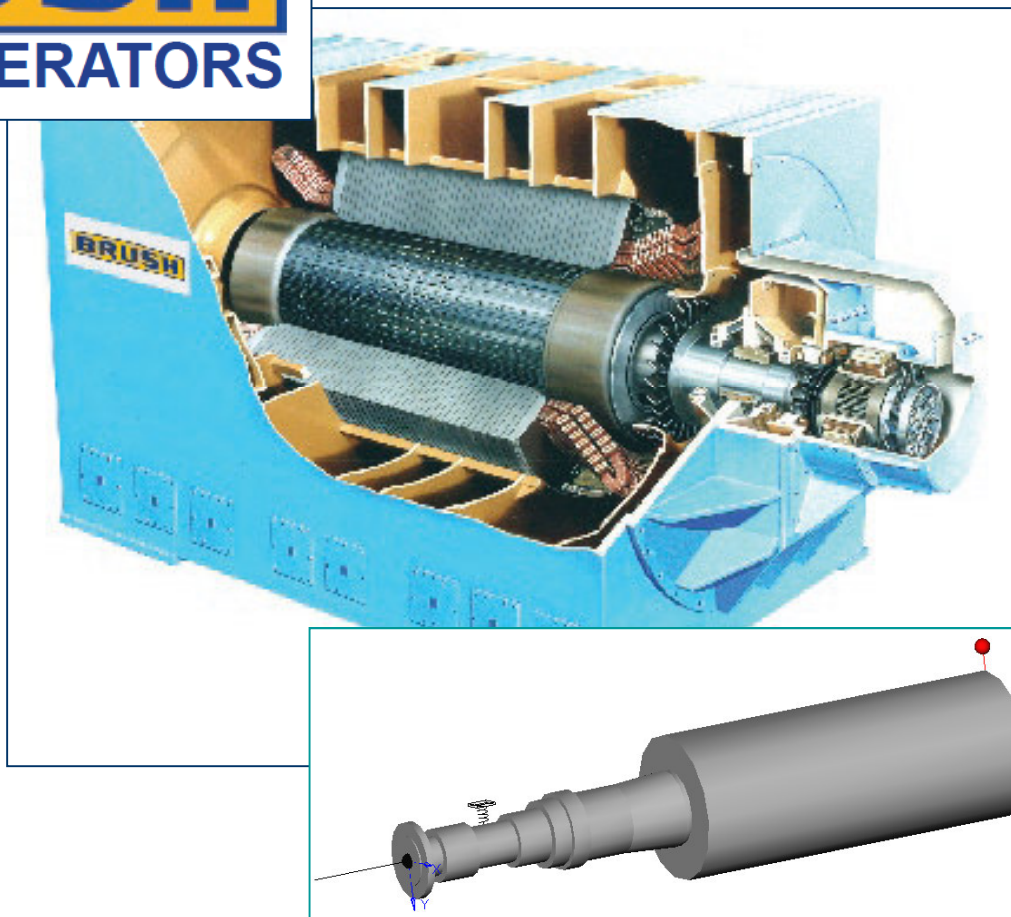
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Generator modeling



BRUSH
TURBOGENERATORS



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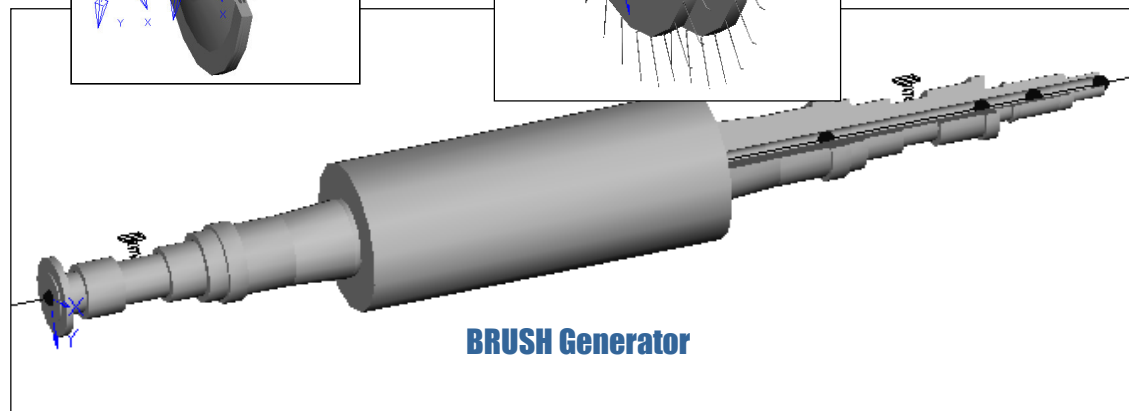
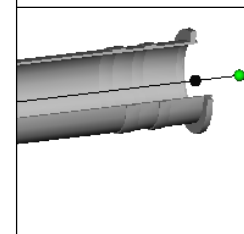
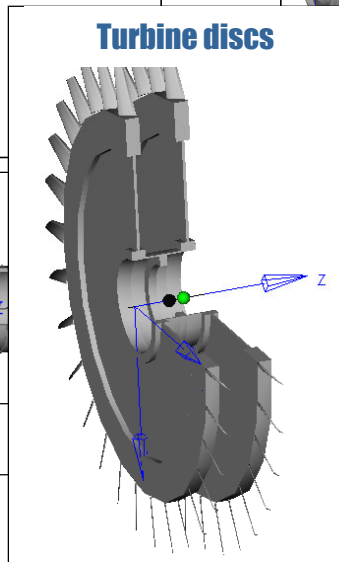
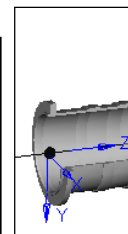
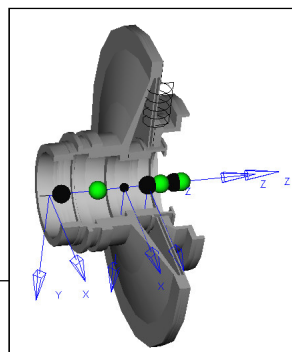
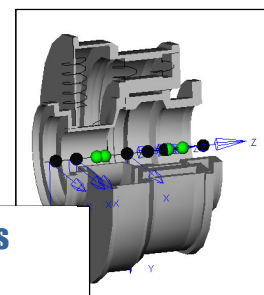
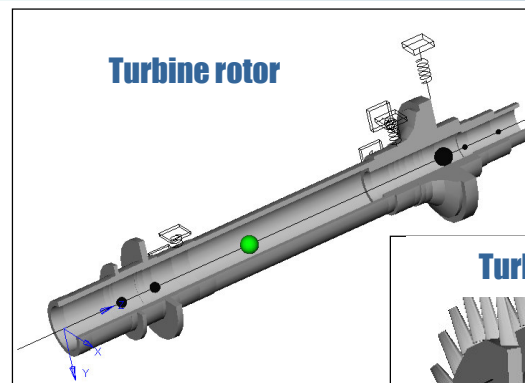
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Modeling

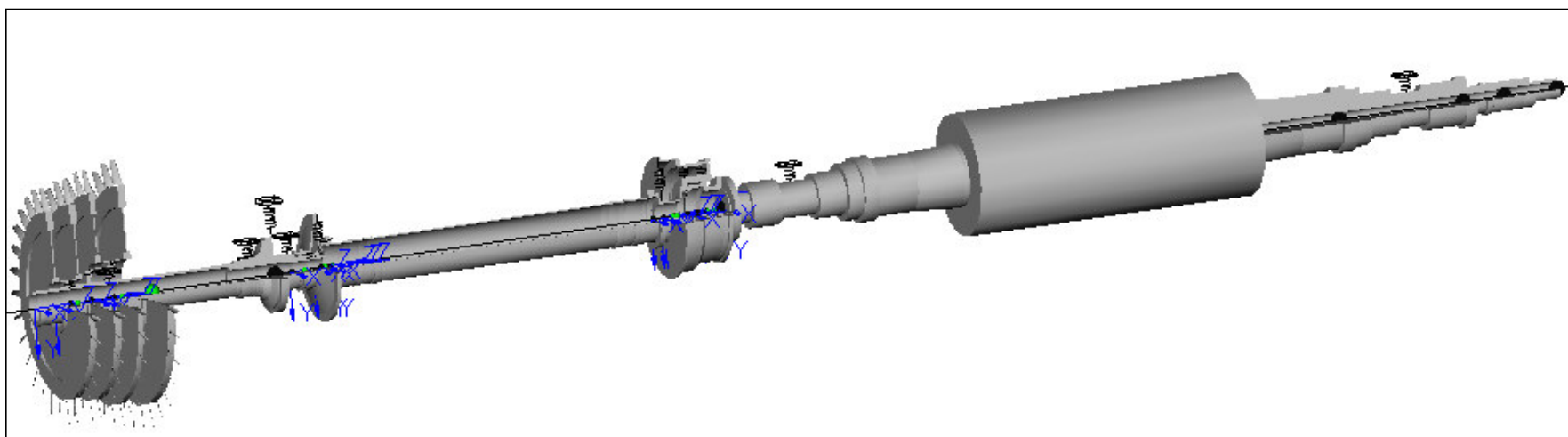
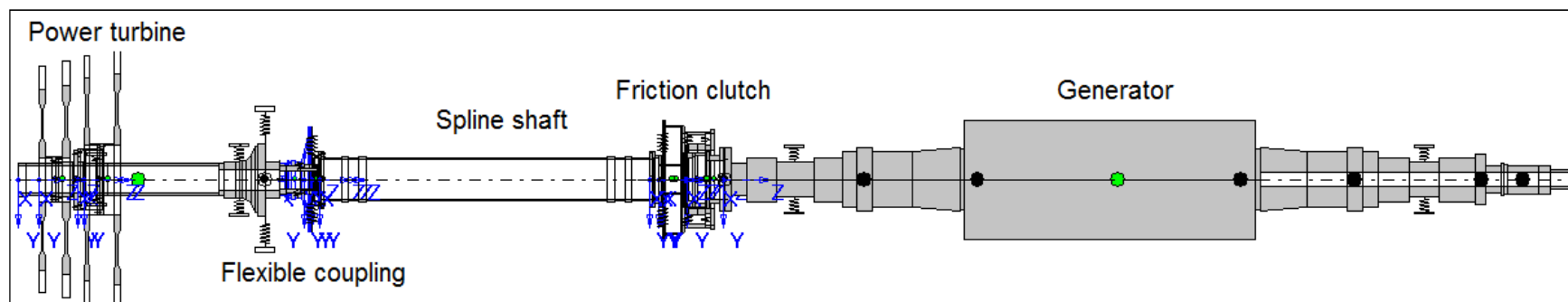
A dynamic model may be multi-level and consists of the following structure items:

- Subsystems – low level structure item constructed out of standard elements
- Assembly – may consist of any structure items
- Sub-models – structure parts stored in an outer file not inside the project

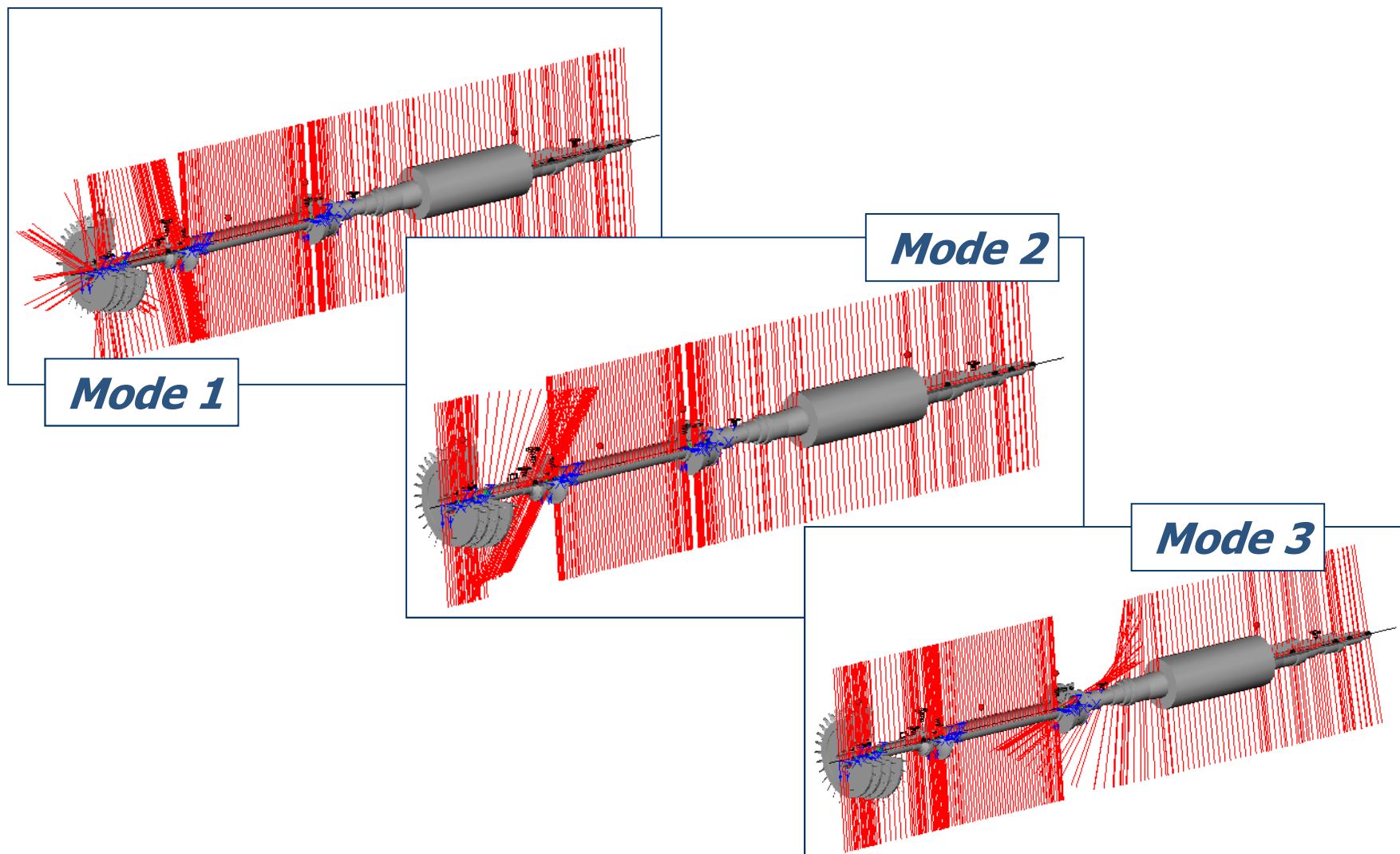
Structure items are assembled with different types of links (bearings)



Complete drive train model



Torsional vibrations of drive train

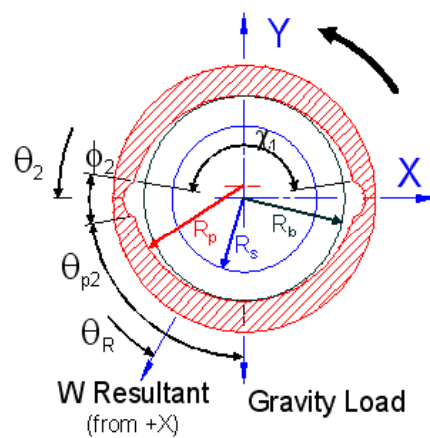


Journal bearings coefficients (*RMT, Inc. s/w*)

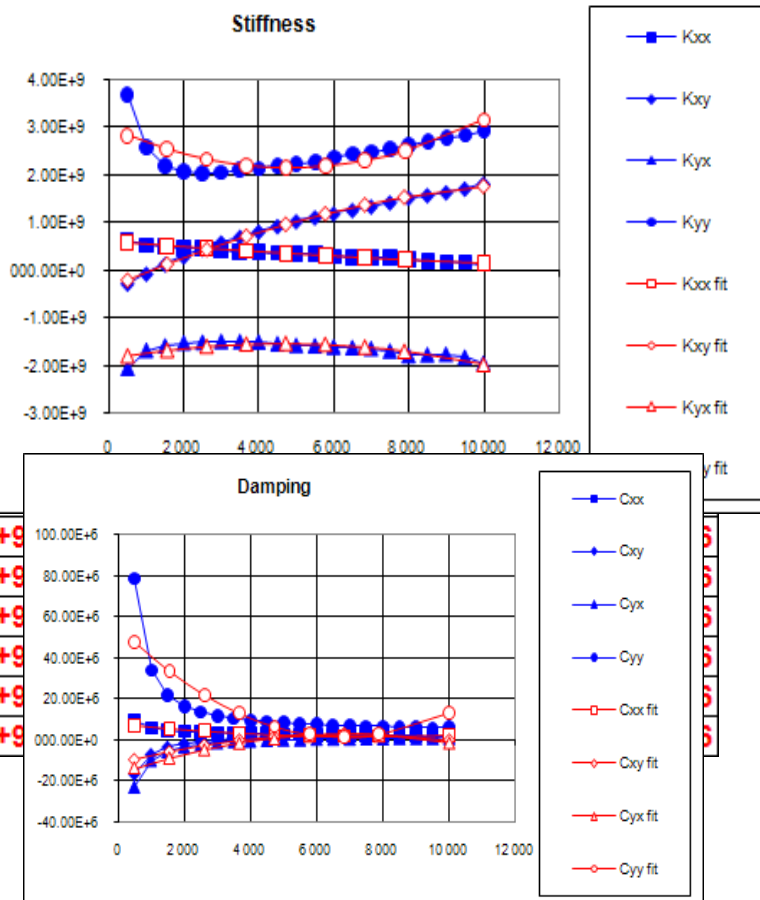


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Speed rpm	Kxx N/m	Kxy N/m	Kyx N/m	Kyy N/m	Cxx N-s/m	Cxy N-s/m	Cyx N-s/m	Cyy N-s/m
500	621.54E+6	-268.71E+6	-2.06E+9	3.67E+9	9.23E+6	-16.38E+6	-23.19E+6	78.66E+6
1 000	527.01E+6	-62.95E+6	-1.68E+9	2.59E+9				
1 500	510.31E+6	135.17E+6	-1.59E+9	2.20E+9				
2 000	471.51E+6	298.15E+6	-1.52E+9	2.08E+9				
				2.03E+9				
				2.05E+9				
				2.10E+9				
				2.13E+9				
				2.18E+9				
				2.23E+9				
				2.27E+9				
				2.37E+9				
				2.44E+9				
				2.48E+9				
				2.54E+9				
				2.63E+9				
				2.70E+9				
9 000	175.69E+6	1.64E+9	-1.76E+9	2.77E+9				
9 500	165.85E+6	1.72E+9	-1.83E+9	2.83E+9				
10 000	126.22E+6	1.82E+9	-1.95E+9	2.94E+9				



χ_1 = #1 Pad Arc Length
 θ_{p2} = #2 Pad Leading Edge to Min. Clearance Angle
 θ_2 = #2 Pad Grv CL Loc. from +X
 ϕ_2 = #2 Pad Grv Arc Length
 θ_R = W Resultant Angle
 $C_b = 2(R_b - R_s)$
 $C_p = 2(R_p - R_s)$
 C_p = Diametral Clearance
 Lobe Preload = $1 - (C_b / C_p)$
 Lobe Offset = θ_p / χ

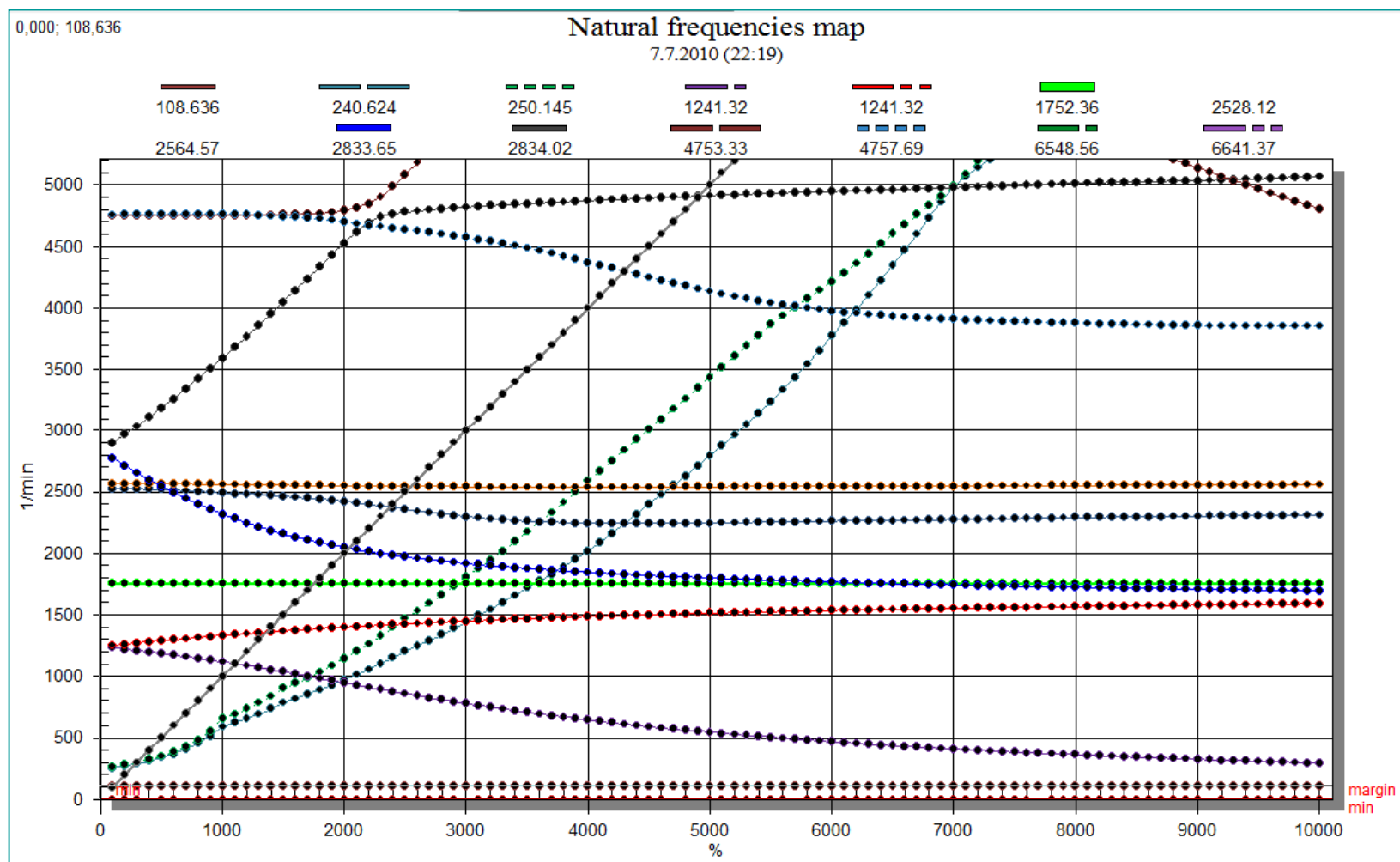


*Data from XLTltpad & XLPocket
can be imported into Dynamics R4*

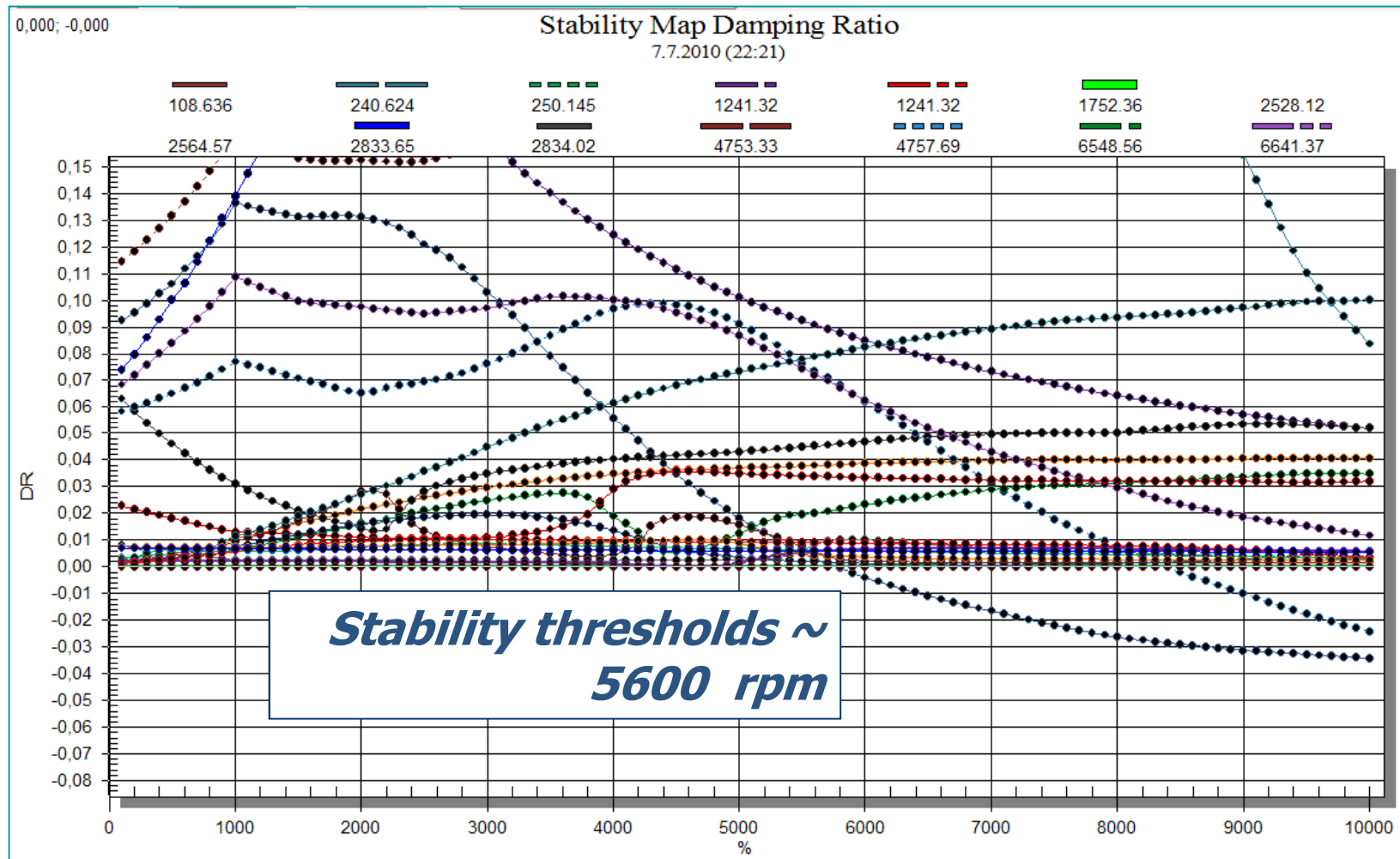
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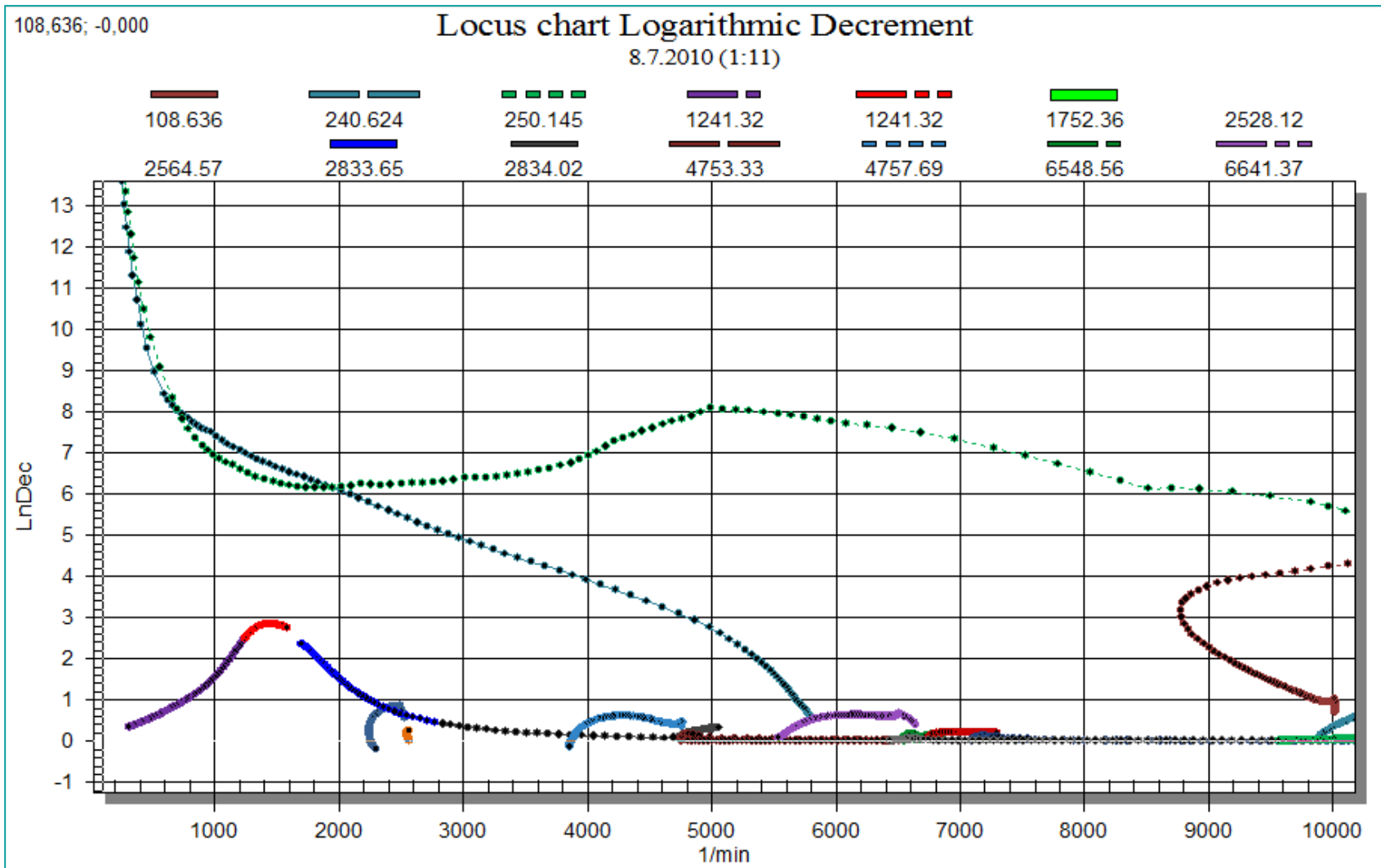
Natural frequency map



Quasi-nonlinear analysis - Stability Map



Locus chart



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Nonlinear analysis

